#### **Marine Black Carbon (BC) Emissions Factors**

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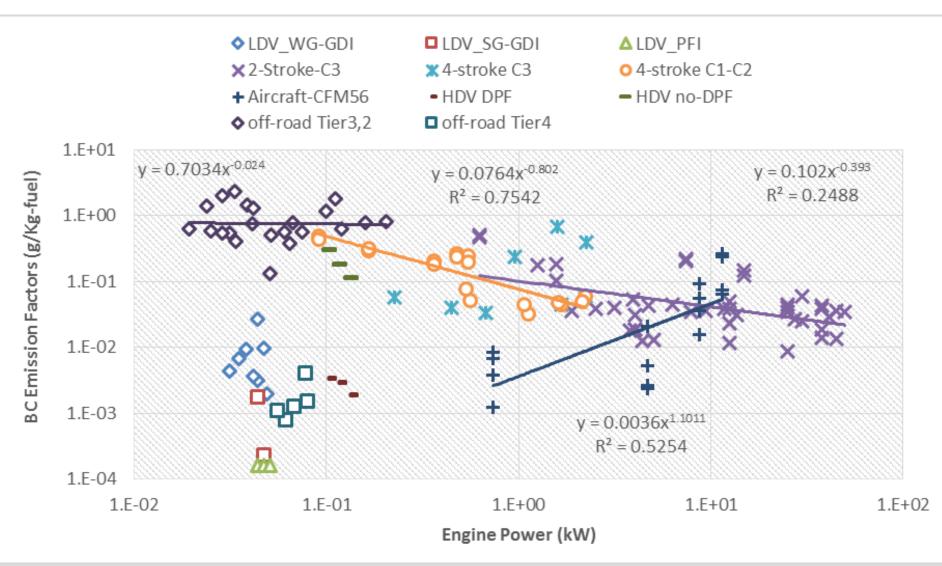


#### **Black Carbon Questions**

- > How do BC emission factor (EF) vary?
  - > Fuels
  - Loads
  - Sources
  - > Dilution
- > Are all BC measurements comparable?

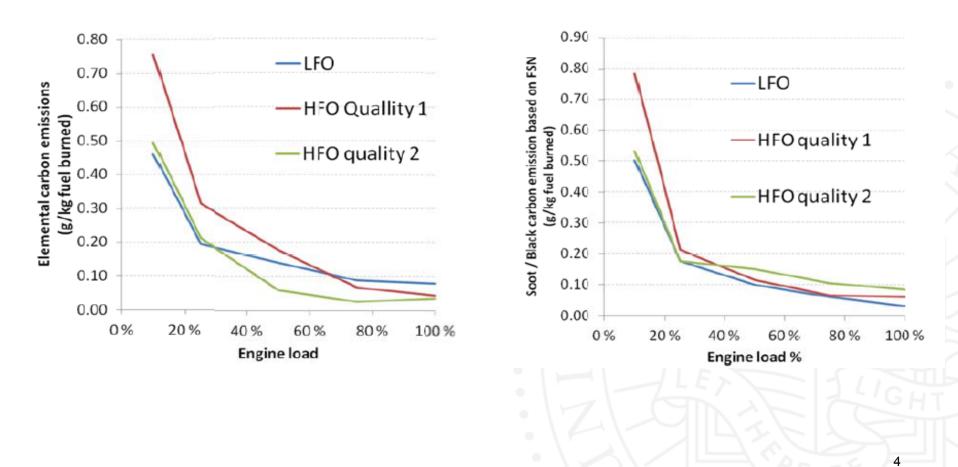


# **EFs Vary by Vocation**





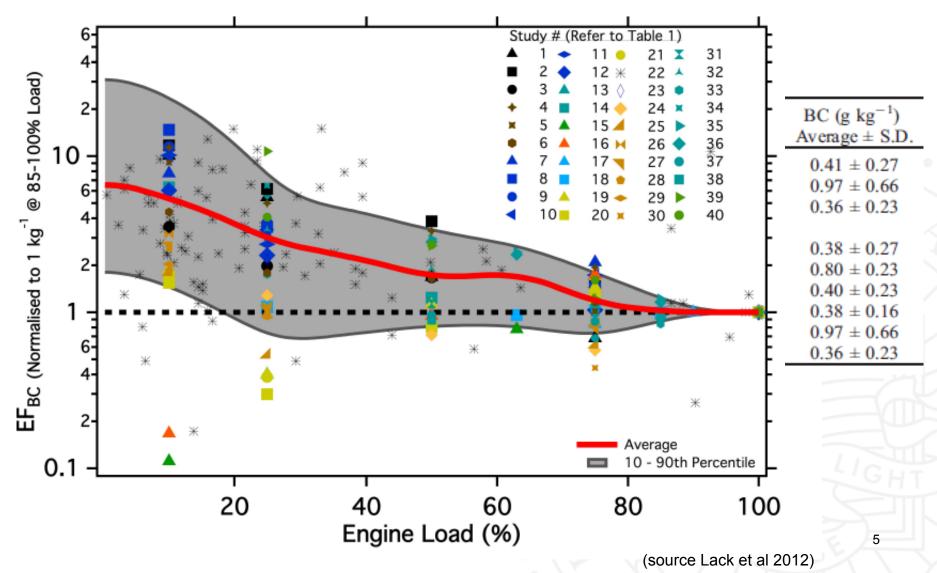
### **EFs Vary with Load Factor**



(source Watsilla)



#### **Plume Studies Show Similar Trend**





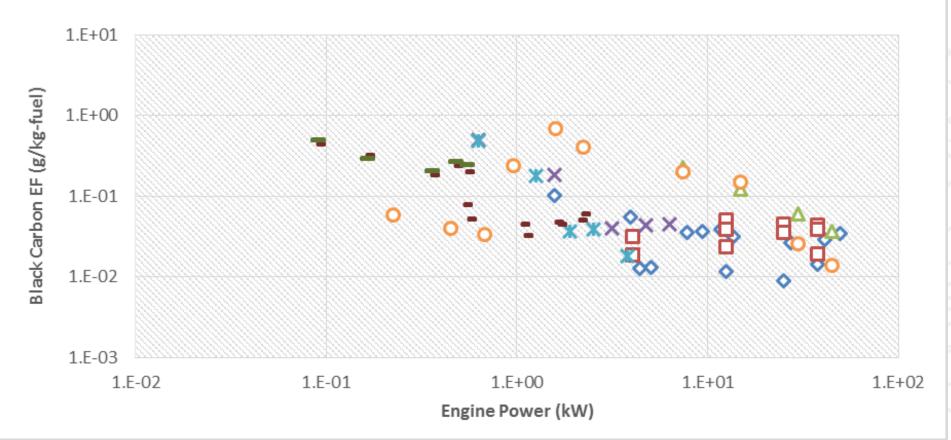
### **EFs Increase with Smaller Size**

♦ 10% □ 25% ▲ 35% × 50% × 75% ○ 100% 50% 100% 10% 1.E+01  $y = 0.1297x^{-0.541}$  $R^2 = 0.7088$  $R^2 = 0.4456$  $R^2 = 0.7383$ Black Carbon EF (g/kg-fuel) 1.E+00 x 0 Ξ 1.E-01 1.E-02 1.E-03 1.E-02 1.E-01 1.E+00 1.E+01 1.E+02 Engine Power (kW)



## **EFs Also Vary with Fuel**







## **Published BC EF Differences**

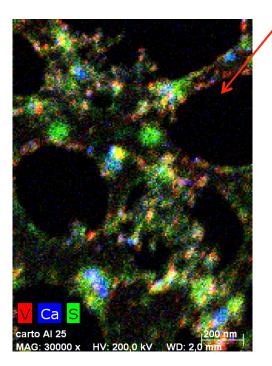
BC emission fuel burned	g/kg fuel	Method of determination
Lack et al	0.36 -1	Optical/photoacoustic
Agrawal et al	0.1	Thermal
Corbett et al	0.37	
Petzold et al	0.179 ± 0.018	Optical
Petzold et al	0.06 (85% load) 0.36 (10% load)	Optical

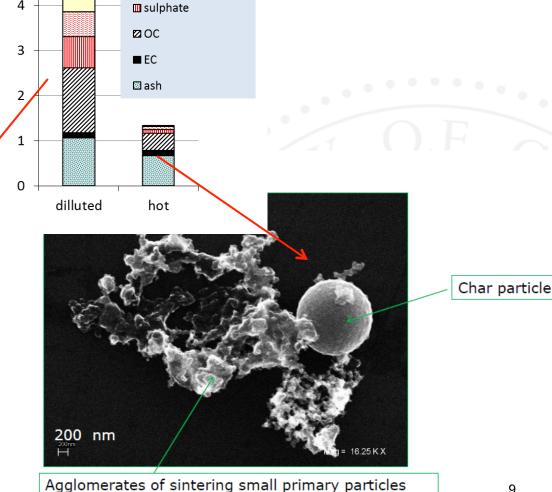
- Variability between reported EF varies by factor of 10
- Measurements varied from optical, thermal, and photo acoustic
- Measurements varied from stack to plume sampling (10:1 vs > 1000:1 dilution)

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#### **PM is Composed of Various** Coatings on BC and it Varies with sulphate Dilution **Ø** 0C 3 EF [g/kg-fuel]





Source: Moldanova et al ICCT 2014 workshop



## Source of BC EF Biases: Measurements Issues?

- Light absorbing carbon measurement methods (filter absorption methods) don't measure BC and need good assumptions about BC mass absorption which varies.
- Laser incandescence (SP2 approach)
  - Loosely packed agglomerates behave as individual particles causing the measurement response differences (Gysel M. et al 2012)

M. Gysel1, M. Laborde1, A. A. Mensah2, J. C. Corbin2, A. Keller3, J. Kim4, A. Petzold5, and B. Sierau2 Technical Note: The single particle soot photometer fails to reliably detect PALAS soot nanoparticles, Atmos. Meas. Tech., 5, 3099–3107, 2012



## Source of BC EF Biases: Measurements Issues?

- Thermal optical methods (NIOSH/IMPROVE) may be influenced by the PM coatings
- Filter Smoke Number/Meter is requires calibration and could vary with total PM concentration and PM coatings.
- Photoacoustic and LLI (laser incandescence) methods are more direct, but require calibration which requires assumptions



### Recommendations

- Quantify/identify possible interferences of measurement methods
  - > As fuel, load, and engine type vary
  - BC measurement systems: LAC types, Thermal optical, and Photo acoustic
- Suggest and recommend BC pre-sample conditioning
  - Thermo-denuder, dilution, temperatures, residence time.



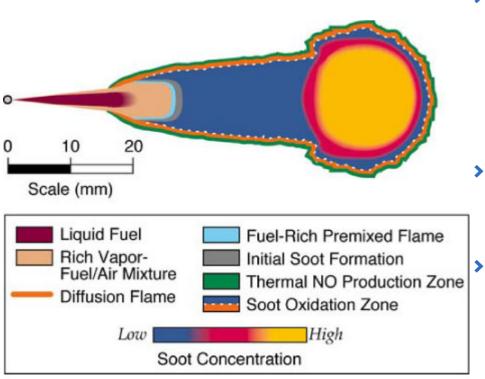
#### Backup



## **Combustion Soot Formation**

#### (Liquid Fuel)

>





- Industry conditions
  - > Engine speed
    - Diesel (3600, 2100, 800, 75 rpm)
    - Gasoline (up to 10,000 rpm)
  - Fuel quality (distillates, residual, sulfur level)

#### **Operational conditions**

 Fuel control issues (engine speed/load vs load only change)

#### **Mitigation and Controls**

- Catalyst, diesel particulate filters (distillates)
- Scrubbers (residual fuels)
- Solid fuel combustion
- Gaseous fuel combustion

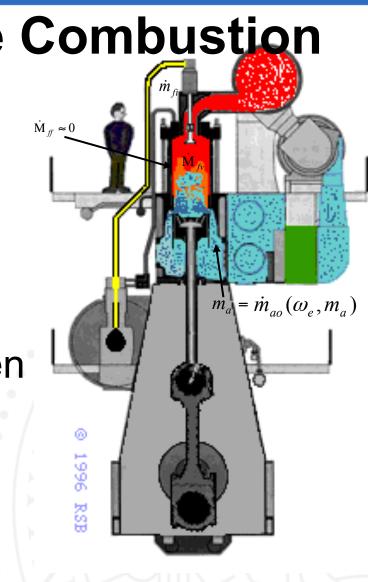


## **Marine 2-Stroke Combustion**

- Cross head type
- Inlet ports uncovered by piston motion, not valves
- Forced scavenging
- Low speed (60-90 rpm) → fuel well mixed with air even with low quality fuels

$$m_a = \dot{m}_{ao}(\omega_e, m_a)$$

$$m_{fi} = \dot{m}_{fv} + \dot{m}_{ff}$$



Large Sulzer 2-stroke cartoon Ref: http://www.reddit.com



 $m_a = \dot{m}_{ao}(\omega_e, m_a)$ 

## 4-Stroke Diesel Combustion

- High power medium RPM range engines (1000 – 2000 rpm)
- No intake air throttling
- Air/fuel injection less dynamic than LDV, but more dynamic than large 2-stroke

$$m_{a} = \dot{m}_{ao}(\omega_{e}, m_{a})$$
$$m_{f} = \dot{m}_{fv} + \dot{m}_{ff}$$



*т*<sub>fi</sub>

 $\dot{M}_{ff}$ 

4-stroke CI. Modified from Ref: World press



# **Spark Ignited Engine Combustion**

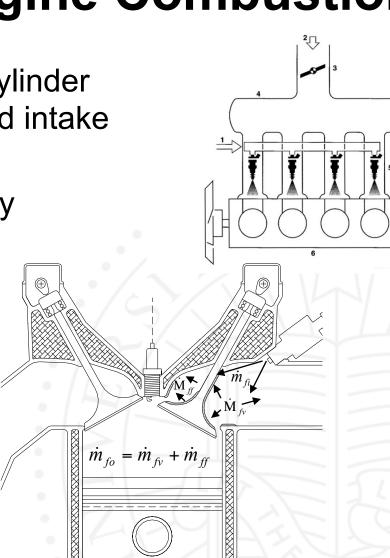
- Fuel injected outside of the cylinder onto the back of the valve and intake manifold
- Complex film and fuel delivery equations

$$\dot{m}_{fv} = (1 - \chi)m_{fi}$$

$$\dot{m}_{ff} = \lambda M_f$$

 $\dot{m}_{fo} = \dot{m}_{fv} + \dot{m}_{ff}$ 

Ref: Bosch and K Johnson 2004



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